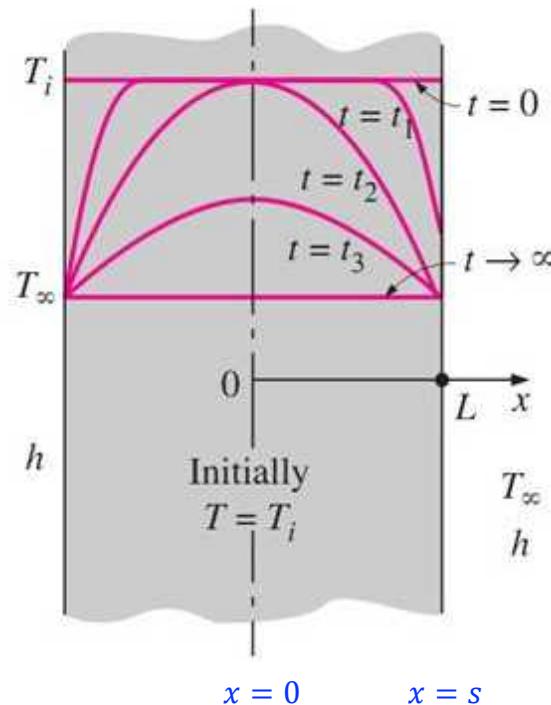


5. Unsteady Heat Conduction (slab)

Assume no heat generation (no heat source)



i. Heat conduction equation

$$\frac{\partial T}{\partial t} = \alpha \frac{\partial^2 T}{\partial x^2} + \frac{\dot{q}}{\rho C_p}$$

I.C. $T(x, 0) = T_a$

B.C. 1) $T(\pm s, t) = T_s$

B.C. 2) $\frac{\partial T(0, t)}{\partial x} = 0$

ii. Solution → Temp. profile

$$\frac{T_s - T}{T_s - T_a} = 2 \sum_{n=0}^{\infty} \frac{(-1)^n}{\left(n + \frac{1}{2}\right)\pi} e^{-\left(n + \frac{1}{2}\right)\pi \left(\frac{\alpha t}{s^2}\right)} \cos \left\{ \left(n + \frac{1}{2}\right)\pi \left(\frac{x}{s}\right) \right\}$$

iii. Mean Temp. $\bar{T}(t)$

$$\frac{T_s - \bar{T}(t)}{T_s - T_a} = \frac{8}{\pi^2} \left\{ e^{-\left(\frac{\pi}{2}\right)^2 \left(\frac{\alpha t}{s^2}\right)} + \frac{1}{9} e^{-9\left(\frac{\pi}{2}\right)^2 \left(\frac{\alpha t}{s^2}\right)} + \frac{1}{25} e^{-25\left(\frac{\alpha t}{s^2}\right)} + \dots \right\}$$

Fourier's number, $Fo, (N_{Fo})$

$$Fo = \frac{\alpha t}{s^2}$$

if $Fo \left(= \frac{\alpha t}{s^2}\right) > 0.1$ then,

$$\frac{T_s - \bar{T}(t)}{T_s - T_a} = \frac{8}{\pi^2} e^{-\left(\frac{\pi}{2}\right)^2 \left(\frac{\alpha t}{s^2}\right)} = \frac{8}{\pi^2} e^{-\left(\frac{\pi}{2}\right)^2 Fo}$$

$$T \Big|_{T_a \rightarrow \bar{T}} \frac{1}{\alpha} \left(\frac{2s}{\pi}\right)^2 \ln \left\{ \frac{8}{\pi^2} \left(\frac{T_s - T_a}{T_s - \bar{T}} \right) \right\}$$

iv. Total heat transferred Q_T

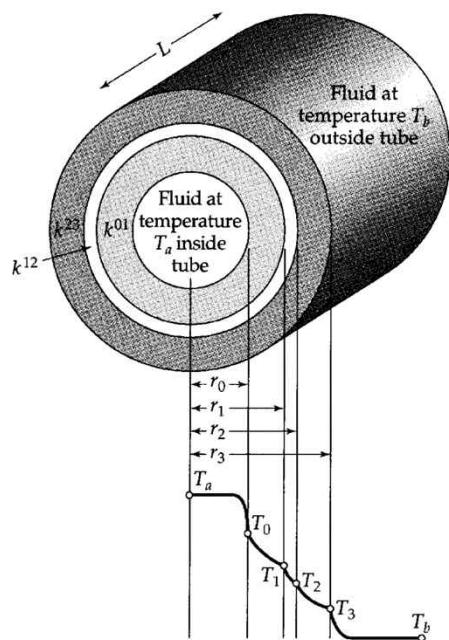
$$Q_T = m C_p (\bar{T} - T_a) = \rho \{A s\} C_p (\bar{T} - T_a)$$

Or total heat transferred per unit area

$$\begin{aligned} Q_T/A &= \frac{\rho \{A s\} C_p (\bar{T} - T_a)}{A} \\ &= \rho s C_p (\bar{T} - T_a) \end{aligned}$$

5. Unsteady Heat Conduction (cylinder)

Assume no heat generation (no heat source)



i. Heat conduction equation

$$\frac{\partial T}{\partial t} = \alpha \frac{\partial}{r \partial r} \left(r \frac{\partial T}{\partial r} \right) + \frac{\dot{q}}{\rho C_p}$$

I.C. $T(r, 0) = T_a$

B.C. 1) $T(R, t) = T_s$

B.C. 2) $\frac{\partial T(0,t)}{\partial r} = 0$

ii. Solution → Temp. profile

$$\frac{T_s - T(r, t)}{T_s - T_a} = \sum_{n=1}^{\infty} 2 e^{-m_i(\frac{\alpha t}{R^2})} \frac{J_o\left[m_i \frac{r}{R}\right]}{m_i J_i(-m_i)}$$

m_i : eigenvalues of Bessel function $J_o(m_i) = 0$

iii. Mean Temp. $\bar{T}(t)$

$$\frac{T_s - \bar{T}(t)}{T_s - T_a} = \sum_{n=1}^{\infty} \frac{4}{m_i^4} e^{-m_i^2 \left(\frac{\alpha t}{R^2} \right)}$$

if $Fo \left(= \frac{\alpha t}{s^2} \right) > 0.1$ then,

Fourier's number, $Fo, (N_{Fo})$

$$Fo = \frac{\alpha t}{s^2}$$

or

$$\frac{T_s - \bar{T}(t)}{T_s - T_a} \cong 0.692 e^{-5.78 \left(\frac{\alpha t}{s^2} \right)} = 0.692 e^{-5.78 Fo}$$

$$t = \frac{R^2}{5.78 \alpha} \ln \frac{T_s - T_a}{T_s - \bar{T}}$$

iv. Total heat transferred Q_T

$$Q_T = m C_p (\bar{T} - T_a) = \rho (\pi R^2 L) C_p (\bar{T} - T_a)$$

Also, total heat transferred per unit area

$$Q_T/A = \frac{\rho (\pi R^2 L) C_p (\bar{T} - T_a)}{2\pi RL}$$

Cf. for slab

$$Q_T/A = \rho s C_p (\bar{T} - T_a)$$

Ex. 10.10) A layer of pulverized cork (insulator)

$$\rho = 900 \text{ kg/m}^3$$

$$k = 0.13 \text{ W/m°C}$$

$$C_p = 1.67 \text{ J/g°C} = 1.67 \times 10^3 \text{ J/kg°C}$$

$$T_s = 121.1^\circ\text{C}$$

$$T_a = 21.1^\circ\text{C}$$

$$s = \frac{2.54}{2} \text{ cm} = 1.27 \text{ cm} = 1.27 \times 10^{-2} \text{ m}$$

$$\alpha \left(\equiv \frac{k}{\rho C_p} \right) = \frac{0.13 \left(\frac{\text{W}}{\text{m°C}} \right)}{900 \left(\frac{\text{kg}}{\text{m}^3} \right) 1.67 \times 10^3 \left(\frac{\text{J}}{\text{kg°C}} \right)} \\ = 8.65 \times 10^{-8} (\text{m}^2/\text{s})$$

$\bar{T} = 98.9^\circ\text{C}$ 라면 그 때의 t ?

$$\frac{T_s - \bar{T}}{T_s - T_a} = \frac{121.1 - 98.9}{121.1 - 21.1} = 0.222$$

$$Fo = 0.52 = \frac{\alpha t}{s^2}$$

$$t = 0.52 \frac{s^2}{\alpha} = 0.52 \frac{(1.27 \times 10^{-2})^2 \text{ m}^2}{8.65 \times 10^{-8} (\text{m}^2/\text{s})} = 969.6 \text{ s}$$

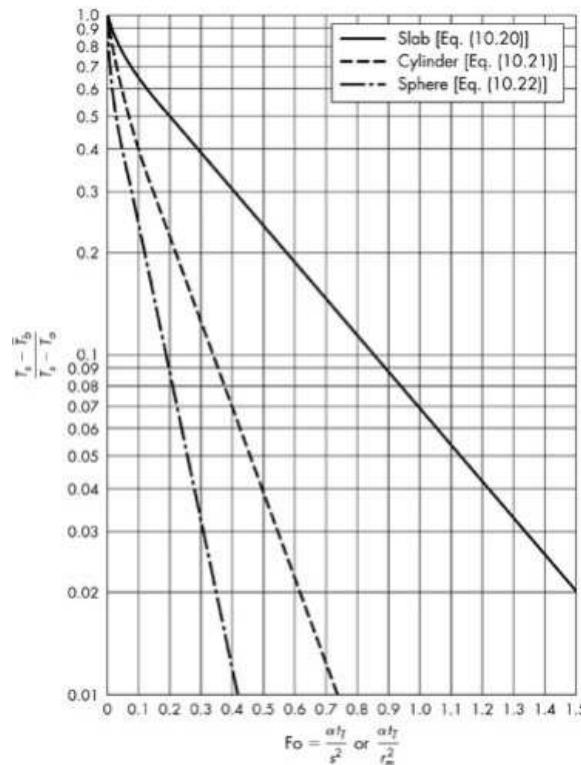


Fig. 10.5에서 y 축의 값이 0.222 면 $Fo \approx 0.52$

$$Q_T = \rho\{A s\}C_p(\bar{T} - T_a)$$

$$\begin{aligned}\frac{Q_T}{A} &= \rho s C_p (\bar{T} - T_a) \\ &= 900 \left(\frac{kg}{m^3} \right) 1.67 \times 10^3 \left(\frac{J}{kg \text{ } ^\circ\text{C}} \right) 1.27 \times 10^{-2} (m)(98.9 - 21.1)^\circ\text{C} \\ &= 1686.43 \text{ } kJ/m^2\end{aligned}$$

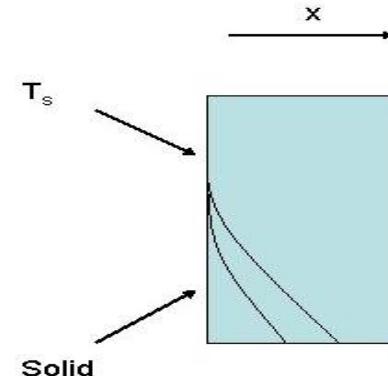
5. Unsteady Heat Conduction

2) Semi-infinite solids (constant surface temperature)

1. Heat Conduction Equation

$$\frac{\partial T}{\partial t} = \alpha \frac{\partial^2 T}{\partial x^2}$$

I.C. $T(x, 0) = T_a$
 B.C. 1) $T(0, t) = T_s$
 B.C. 2) $T(\infty, t) = T_a$



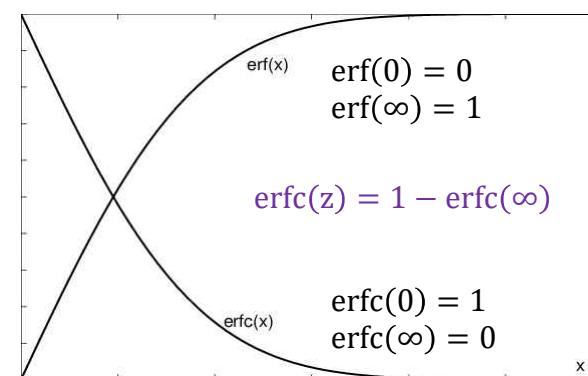
2. Solution → Temperature distribution (profile)

$$\frac{T_s - T}{T_s - T_a} = \frac{2}{\sqrt{\pi}} \int_0^z e^{-z^2} dz$$

where, $z = \frac{x}{2\sqrt{\alpha t}}$

Error function (Gauss Error Integration)

$$\text{erf}(z) = \frac{2}{\sqrt{\pi}} \int_0^z e^{-z^2} dz$$



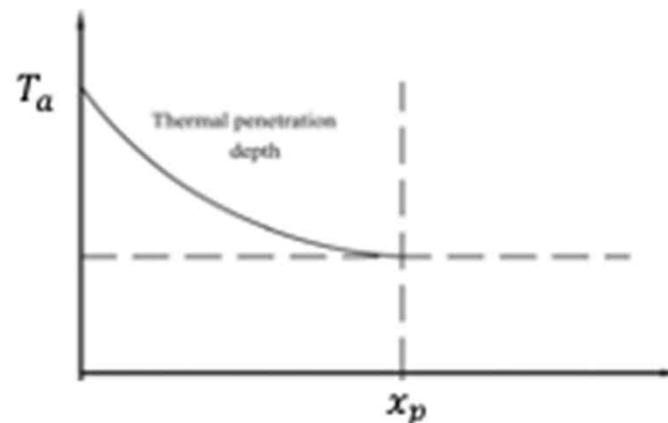
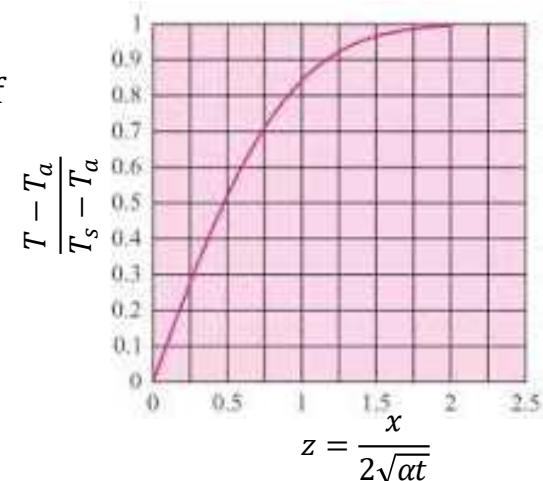
5. Unsteady Heat Conduction

3) Penetration depth, x_p

Distance from the surface at which the temperature change is 1% of the initial change in the surface change

$$\frac{T - T_a}{T_s - T_a} = 0.01$$

$$\frac{T_s - T}{T_s - T_a} = 0.99$$



$$\begin{aligned} \text{erf}(z) &= 0.99 \text{ at } z = 1.82 \rightarrow (z_p) \\ \text{i.e. } z_p &= \frac{x_p}{2\sqrt{\alpha t}} = 1.82 \\ x_p &= 3.64\sqrt{\alpha t} \end{aligned}$$

5. Unsteady Heat Conduction

4) Total heat transferred, Q_T

$$Q_T = \int_0^t q \Big|_{x=0} dt = \int_0^t \left[-kA \frac{dT}{dx} \right]_{x=0} dt$$

$$\Rightarrow \frac{Q_T}{A} \int_0^t \left[-kA \frac{dT}{dx} \right]_{x=0} dt$$

$$\frac{dx}{dz} \frac{dT}{dz} = -\frac{2}{\sqrt{\pi}} (T_s - T_a) e^{-z^2}$$

$$\frac{dx}{dz} \frac{dT}{dx} = -\frac{2}{\sqrt{\pi}} (T_s - T_a) e^{-z^2}$$

$$\frac{dT}{dx} \Big|_{x=0} = \left(\frac{dz}{dx} \right) \left[-\frac{2}{\sqrt{\pi}} (T_s - T_a) e^{-z^2} \right] \Big|_{z=0} = -\frac{T_s - T_a}{\sqrt{\pi} \alpha t}$$

$$\frac{Q_T}{A} = k \frac{T_s - T_a}{\sqrt{\pi} \alpha} \int_0^t \frac{1}{\sqrt{t}} dt$$

$$= 2k(T_s - T_a) \sqrt{\frac{t}{\pi \alpha}} = 2k(T_s - T_a) \sqrt{\frac{\rho C_p k}{\pi}} t$$

Ex. 10.7)

$$T_a = 5^\circ\text{C} \quad \overrightarrow{48 \text{ hr.}} \quad T_s = -20^\circ\text{C}$$

$$\alpha = 0.0011 \text{ m}^2/\text{hr}$$

a) $x|_{T=0^\circ\text{C}} = ?$

$$\frac{T_s - T}{T_s - T_a} (= \operatorname{erf}(z)) = \frac{(-20) - 0}{(-20) - 5} = 0.8 \quad \Rightarrow \quad z \left(= \frac{x}{2\sqrt{\alpha t}} \right) = 0.91$$

From fig. 10.10

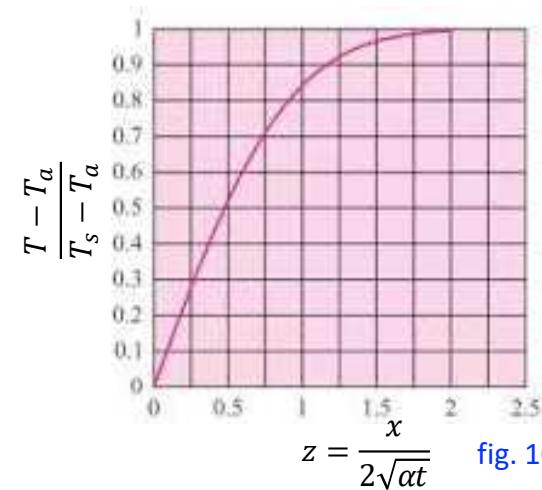


fig. 10.10

$$\Rightarrow x = 0.91 \times 2\sqrt{\alpha t} = 0.91 \times 2\sqrt{0.0011(\text{m}^2/\text{hr}) \times 48(\text{hr})} = 0.418 \text{ m}$$

a) $x_p = ? \quad \frac{T_s - T}{T_s - T_a} = 0.99 \Rightarrow T = 4.75^\circ\text{C} \quad x_p|_{T=4.75^\circ\text{C}}$

$$x_p = 3.64\sqrt{\alpha t} = 3.64\sqrt{0.0011(\text{m}^2/\text{hr}) \times 48(\text{hr})} = 0.836 \text{ m}$$

Problems good to solve : (Probs.) 10.1, 10.2, 10.4, 10.5, 10.6, 10.7 and 10.10