

14.4 Nonblack Surfaces

- Gray surfaces

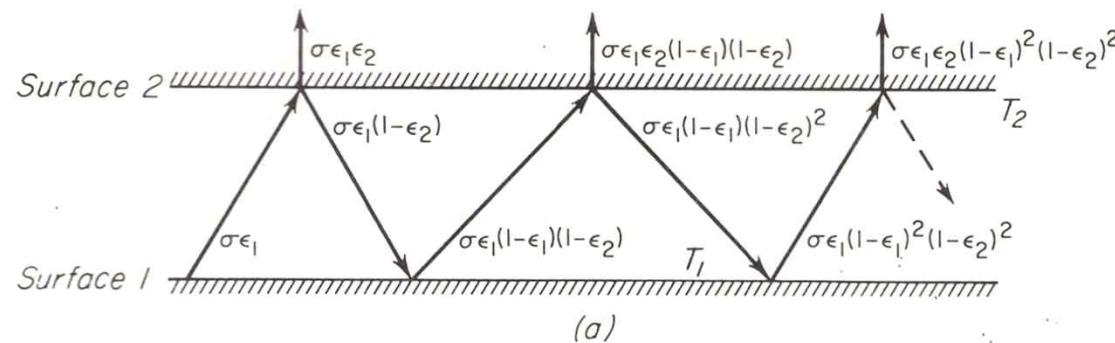
$$q_{12} = \sigma A_1 \mathcal{F}_{12} (T_1^4 - T_2^4) = \sigma A_2 \mathcal{F}_{21} (T_1^4 - T_2^4)$$

where, $\mathcal{F}_{12}, \mathcal{F}_{21}$: overall interchange factor, $f(\epsilon_1, \epsilon_2)$,

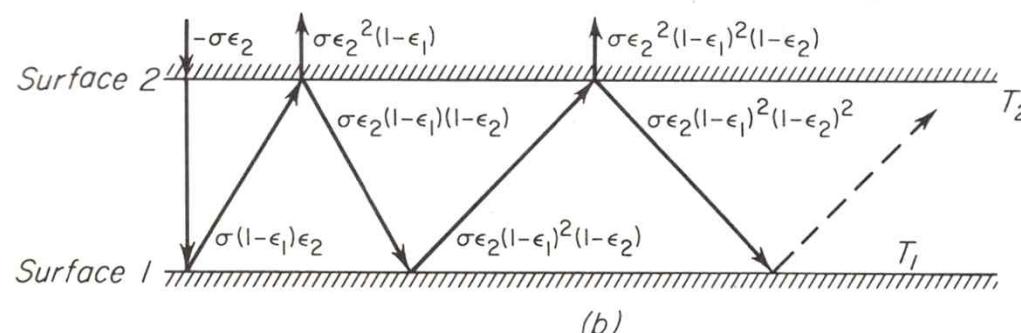
$$\sigma = 5.672 \times 10^{-8} \frac{\text{W}}{\text{m}^2 \cdot \text{K}^4} = 0.1713 \times 10^{-8} \frac{\text{BTU}}{\text{ft}^2 \cdot \text{hr} \cdot {}^\circ\text{F}^4}$$

14.4 Nonblack Surfaces

- Gray surfaces



(a)



(b)

14.4 Nonblack Surfaces

$$(a) q_{1 \rightarrow 2} = \sigma T_1^4 \epsilon_1 \epsilon_2 [1 + (1 - \epsilon_1)(1 - \epsilon_2) + (1 - \epsilon_1)^2(1 - \epsilon_2)^2 + \dots]$$

$$(b) q_{2 \rightarrow 1} = [-\sigma \epsilon_2 + \sigma \epsilon_2^2 (1 - \epsilon_1) + \sigma \epsilon_2 (1 - \epsilon_1)^2 (1 - \epsilon_2)^2 + \dots] T_2^4$$

$$= -\sigma T_2^4 [\epsilon_2 - \epsilon_2^2 (1 - \epsilon_1) + \sigma \epsilon_2^2 (1 - \epsilon_1)^2 (1 - \epsilon_2)^2 - \dots]$$

$$= -\sigma T_2^4 \{ \epsilon_2 - \epsilon_2^2 (1 - \epsilon_1) [1 + (1 - \epsilon_1)(1 - \epsilon_2) + (1 - \epsilon_1)^2(1 - \epsilon_2)^2 + \dots] \}$$

14.4 Nonblack Surfaces

$$q_{12} = (a) + (b) = q_{1 \rightarrow 2} + q_{2 \rightarrow 1}$$

let $y = (1 - \epsilon_1)(1 - \epsilon_2)$,

$$\begin{aligned} q_{12} &= \sigma T_1^4 \epsilon_1 \epsilon_2 (1 + y + y^2 + y^3 + \dots) \\ &\quad - \sigma T_2^4 [\epsilon_2 - \epsilon_2^2 (1 - \epsilon_1) (1 + y + y^2 + y^3 + \dots)] \end{aligned}$$

$$\text{since } y < 1 \text{ then, } 1 + y + y^2 + y^3 + \dots = \frac{1}{1-y}$$

$$q_{12} = \sigma T_1^4 \epsilon_1 \epsilon_2 \frac{1}{1-y} - \sigma T_2^4 \left[\epsilon_2 - \epsilon_2^2 (1 - \epsilon_1) \frac{1}{1-y} \right]$$

$$q_{12} = \sigma A_1 \mathcal{F}_{12} (T_1^4 - T_2^4) = \sigma A_2 \mathcal{F}_{21} (T_1^4 - T_2^4) \text{ 0이므로,}$$

$$\mathcal{F}_{12} = \frac{1}{\frac{1}{\epsilon_1} + \frac{1}{\epsilon_2} - 1}$$

14.5 Total heat transfer by conduction, convection, and radiation

$$\frac{q_T}{A} = \frac{q_c}{A} + \frac{q_r}{A} = h_c(T_w - T) + \sigma\epsilon_w(T_w^4 - T^4)$$

where, $\frac{q_T}{A}$: total heat flux, $\frac{q_c}{A}$: heat flux by conduction and convection

$\frac{q_r}{A}$: heat flux by radiation, h_c : convective heat transfer coefficient

ϵ_w : emissivity of surface, T_w : temperature of surface, T : surrounding Temp.

$$\frac{q_T}{A} = (h_c + h_r)(T_w - T)$$

h_r : radiation heat transfer coefficient

$$h_r \equiv \frac{q_r}{A(T_w - T)}$$

Example

1. 두 강철판이 평행하게 서로 마주보고 있다. 두 강철판은 온도차이가 있으며, 한쪽의 온도는 986 K이고 나머지 한 쪽의 온도는 478 K이다. 그리고 더운 쪽과 덜 더운 쪽의 방사율은 각각 0.7, 0.4이다. 다음에 답하시오 (Boltzmann상수= $5.672 \times 10^{-8} \text{ W/m}^2 \cdot \text{K}^4$)

- (a) 두 강철판 사이에 전해지는 heat flux는?
- (b) 두 강철판 사이에 같은 크기의 얇은 검은칠을 한 알루미늄판을 나란히 놓는다면, 그 알루미늄 판의 온도는 몇 °C가 되겠는가?

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$$\epsilon_1 = 0.7$$

986 K

$$\epsilon_2 = 0.4$$

478 K

Example

- 가정 :
 - 두 강철판은 gray body이
 - 두 강철판의 면적은 같다.
 - 강철판 사이에 재복사되는 (reradiated) 내화벽 (refractory wall)은 없다

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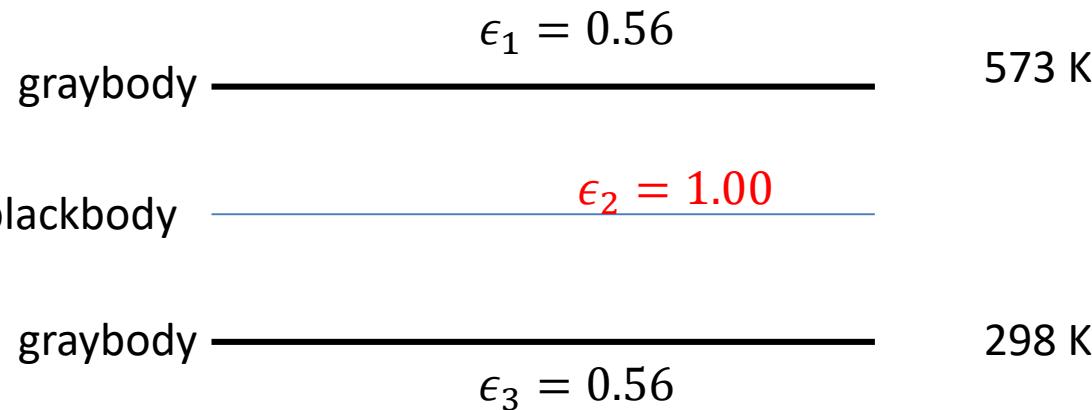
a) $q/A = \sigma F_{12} (T_1^4 - T_2^4)$

$$F_{12} = \frac{1}{1/\epsilon_1 + 1/\epsilon_2 - 1} = \frac{1}{1/0.7 + 1/0.4 - 1} = 0.34$$
$$q/A = (5.672 \times 10^{-8}) \cdot 0.34 \cdot (986^4 - 478^4) = 17220.56 \text{ W/m}^2$$

b) 얇은 알루미늄판을 black body로 가정하면 $\epsilon_a = 1$

$$F_{1a} = \frac{1}{1/\epsilon_1 + 1/\epsilon_a - 1} = \frac{1}{1/0.7 + 1/1 - 1} = 0.7$$
$$F_{a2} = \frac{1}{1/\epsilon_a + 1/\epsilon_2 - 1} = \frac{1}{1/1 + 1/0.4 - 1} = 0.4$$
$$\sigma A F_{1a} (T_1^4 - T_a^4) = \sigma A F_{a2} (T_a^4 - T_2^4)$$
$$T_a = \sqrt[4]{\frac{\mathcal{F}_{1a} T_1^4 + \mathcal{F}_{a2} T_2^4}{\mathcal{F}_{1a} + \mathcal{F}_{a2}}} = \sqrt[4]{\frac{0.7 \times 986^4 + 0.4 \times 478^4}{0.7 + 0.4}} = 887.52 \text{ K} = 614.52^\circ\text{C}$$

Example 14.1



a) $T_2 = ?$

2 is blackbody $\rightarrow \epsilon_2 = 1.00$

$$q_{12} = q_{23} \quad \therefore 2 \text{ is blackbody}$$

$$q_{12} = \sigma A_1 \mathcal{F}_{12} (T_1^4 - T_2^4)$$

$$q_{23} = \sigma A_2 \mathcal{F}_{23} (T_2^4 - T_3^4)$$

$$\sigma A_1 \mathcal{F}_{12} (T_1^4 - T_2^4) = \sigma A_2 \mathcal{F}_{23} (T_2^4 - T_3^4)$$

$$\mathcal{F}_{12} = \frac{1}{\frac{1}{\epsilon_1} + \frac{1}{\epsilon_2} - 1} = \frac{1}{\frac{1}{0.56} + \frac{1}{1.0} - 1} = 0.56$$

$$\mathcal{F}_{23} = \frac{1}{\frac{1}{\epsilon_2} + \frac{1}{\epsilon_3} - 1} = \frac{1}{\frac{1}{1.0} + \frac{1}{0.56} - 1} = 0.56$$

$$T_1^4 - T_2^4 = T_2^4 - T_3^4$$

$$T_2 = \sqrt[4]{\frac{T_1^4 + T_3^4}{2}} = 490 \text{ K}$$

Example 14.1

b) Heat flux ?

$$\begin{aligned} q_{12} &= \sigma A_1 \mathcal{F}_{12} (T_1^4 - T_2^4) \\ q_{12}/A &= \sigma \mathcal{F}_{12} (T_1^4 - T_2^4) \\ &= (5.672 \times 10^{-8}) [\text{W/m}^2 \cdot \text{K}^4] \cdot 0.56 \cdot (573^4 - 490.41^4) [\text{K}] \\ &= 1586.85 \text{ W/m}^2 \end{aligned}$$

$$\begin{aligned} q_{23} &= \sigma A_2 \mathcal{F}_{23} (T_2^4 - T_3^4) \\ &= (5.672 \times 10^{-8}) [\text{W/m}^2 \cdot \text{K}^4] \cdot 0.56 \cdot (790^4 - 298^4) [\text{K}] \\ &= 1586.73 \text{ W/m}^2 \end{aligned}$$

$$q_{12} \approx q_{23}$$

*) 만약 2 가 없다면,

$$\begin{aligned} q_{13}/A &= \sigma A_1 \mathcal{F}_{13} (T_1^4 - T_3^4) \\ \mathcal{F}_{13} &= \frac{1}{1/\epsilon_2 + 1/\epsilon_3 - 1} = \frac{1}{1/0.56 + 1/1.0 - 1} = 0.39 \end{aligned}$$

$$\begin{aligned} \frac{q_{13}}{A} &= \sigma A_1 \mathcal{F}_{13} (T_1^4 - T_3^4) = \\ &= (5.672 \times 10^{-8}) [\text{W/m}^2 \cdot \text{K}^4] \cdot 0.39 \cdot (573^4 - 298^4) [\text{K}] \\ &= 2210.17 \text{ W/m}^2 \end{aligned}$$